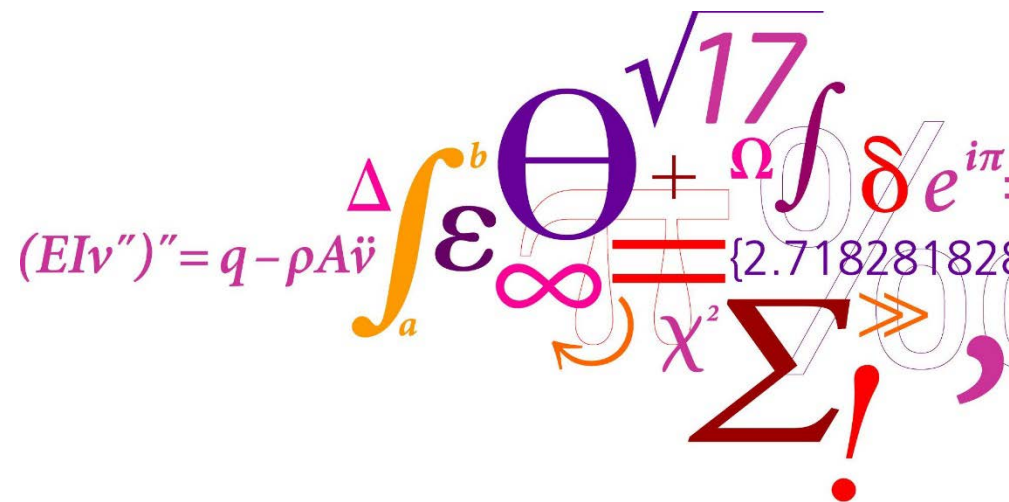


A Fully-Coupled Approach for Modelling Plastic Deformation and Liquid Lubrication in Metal Forming

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Outline

- Theoretical & Numerical Background
- Results & Discussion
- Summary

- **Theoretical & Numerical Background**
 - Equilibrium Equations
 - FE Discretization
 - Viscous Flow within Flow Formulation
- Results & Discussion
- Summary

Equilibrium Equation

In fluid mechanics differential equation of motion is

$$\frac{\partial \sigma'_{ij}}{\partial x_j} - \frac{\partial p}{\partial x_i} + \rho g_i = \rho \left(\frac{\partial u_i}{\partial t} + u_j \frac{\partial u_i}{\partial x_j} \right)$$

↓

viscous
forces

↓

pressure

↓

gravity

↓

inertia term

where $\sigma_{ij} = \sigma'_{ij} - p$ and $p = -\sigma_m$.

For Newtonian fluid,

$$\tau_{xy} = \mu \left(\frac{\partial u}{\partial y} + \frac{\partial v}{\partial x} \right) \qquad \sigma'_{xx} = 2\mu \frac{\partial u}{\partial x}$$

Equilibrium Equation

If the space-dependent acceleration term of Navier-Stokes equation is neglected, the equation reduces to

$$\mu \frac{\partial^2 u_i}{\partial x_j^2} - \frac{\partial p}{\partial x_i} + \rho g_i = \rho \frac{du_i}{dt}$$

$$\sigma_{ij} = \sigma'_{ij} + \sigma_m = \sigma'_{ij} - p$$

which gives the dynamic equilibrium equation of solid mechanics that is used in the flow formulation.

$$\frac{\partial \sigma_{ij}}{\partial x_j} + b_i = \rho a_i \quad b_i = \rho g_i$$

$$a_i = \frac{Du_i}{Dt} = \dot{u}_i + (u_j \cdot \nabla)u_i \cong \dot{u}_i$$

The Flow Formulation

Similarities of viscous flow with flow formulation

- viscous flow formulation

$$\mu \frac{\partial^2 u_i}{\partial x_j^2} - \frac{\partial p}{\partial x_i} + \rho g_i = \rho \frac{du_i}{dt}$$

- identical separation between deviatoric and volumetric terms
- the analogy for 'shear viscosity' and 'bulk viscosity' to ensure the incompressibility constraint of the viscous fluids.

$$\dot{\rho} + \nabla(\rho \mathbf{u}) = 0 \quad \xrightarrow{\partial \rho / \partial t \approx 0} \quad \nabla \mathbf{u} = 0 \quad \rightarrow \quad \dot{\epsilon}_v = 0$$

$$\sigma_m = K \dot{\epsilon}_v \qquad \sigma_m = -p = \mu_v \dot{\epsilon}_v$$

$$\sigma'_{ij} = \frac{2}{3} \frac{\bar{\sigma}}{\dot{\epsilon}} \dot{\epsilon}'_{ij} \qquad \sigma'_{ij} = 2\mu \dot{\epsilon}'_{ij}$$

Finite Element Formulation

Weak Form of Equilibrium Equation

$$\delta\Pi = \int_V \sigma'_{ij} \delta\dot{\epsilon}'_{ij} dV + K \int_V \dot{\epsilon}_V \delta\dot{\epsilon}_V dV - \int_{S_T} t_i \delta u_i dS = 0$$

$$\sum_{m=1}^M \left\{ \int_{V^m} \delta \mathbf{v}^T \mathbf{K}_\mu \mathbf{v} dV^m + K^m \int_{V^m} \delta \mathbf{v}^T \mathbf{C}^T \mathbf{B} \mathbf{v} \mathbf{C}^T \mathbf{B} dV^m - \int_{S_T^m} \delta \mathbf{v}^T \mathbf{N} \mathbf{T} dS^m \right\} = 0$$

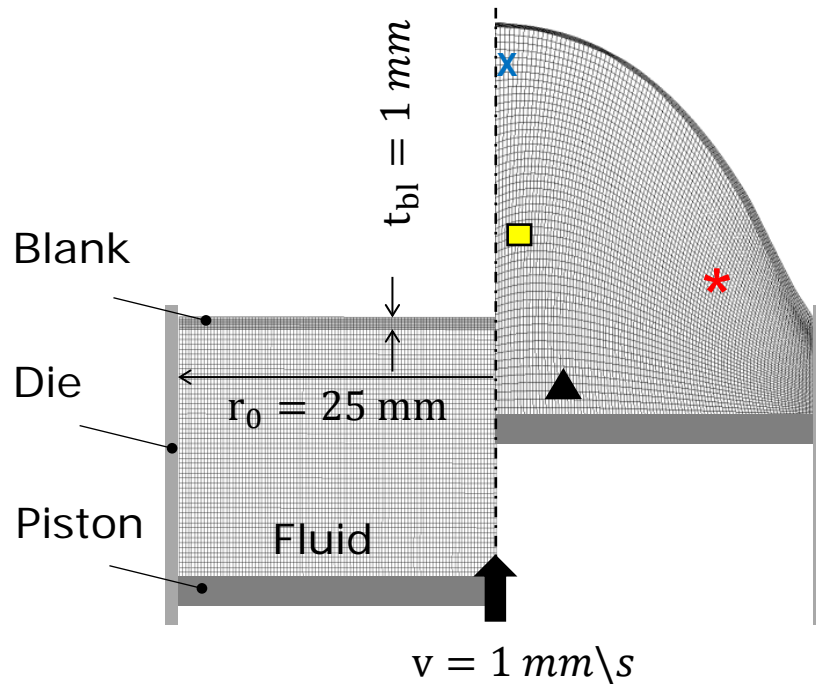
$$\mathbf{K}_\mu = \mathbf{B}^T \mathbf{D}_\mu \mathbf{B} \quad \mathbf{C}^T = [1 \ 1 \ 1 \ 0 \ 0 \ 0] \quad \mathbf{B} = \mathbf{L} \mathbf{N}^T \quad K^m = \mu_v^m = k_0 - k_1 \cdot \frac{\sigma_{ii}}{3}$$

$$\mathbf{D}_\mu = \begin{bmatrix} 2\mu & 0 & 0 & 0 & 0 & 0 \\ 0 & 2\mu & 0 & 0 & 0 & 0 \\ 0 & 0 & 2\mu & 0 & 0 & 0 \\ 0 & 0 & 0 & \mu & 0 & 0 \\ 0 & 0 & 0 & 0 & \mu & 0 \\ 0 & 0 & 0 & 0 & 0 & \mu \end{bmatrix} \quad \mathbf{L} = \begin{bmatrix} \frac{\partial}{\partial x} & 0 & 0 \\ 0 & \frac{\partial}{\partial y} & 0 \\ 0 & 0 & \frac{\partial}{\partial z} \\ \frac{\partial}{\partial y} & \frac{\partial}{\partial x} & 0 \\ 0 & \frac{\partial}{\partial z} & \frac{\partial}{\partial y} \\ \frac{\partial}{\partial z} & 0 & \frac{\partial}{\partial x} \end{bmatrix} \quad \mathbf{N}^T = \begin{bmatrix} N_1 & 0 & 0 & \dots & N_8 & 0 & 0 \\ 0 & N_1 & 0 & \dots & 0 & N_8 & 0 \\ 0 & 0 & N_1 & \dots & 0 & 0 & N_8 \end{bmatrix}$$

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- **Results & Discussion**
 - Bulge Test
 - Lubrication Breakdown
 - Drawing of Sheet with Surface Pockets
 - Deep Drawing
- Summary

Bulge Test

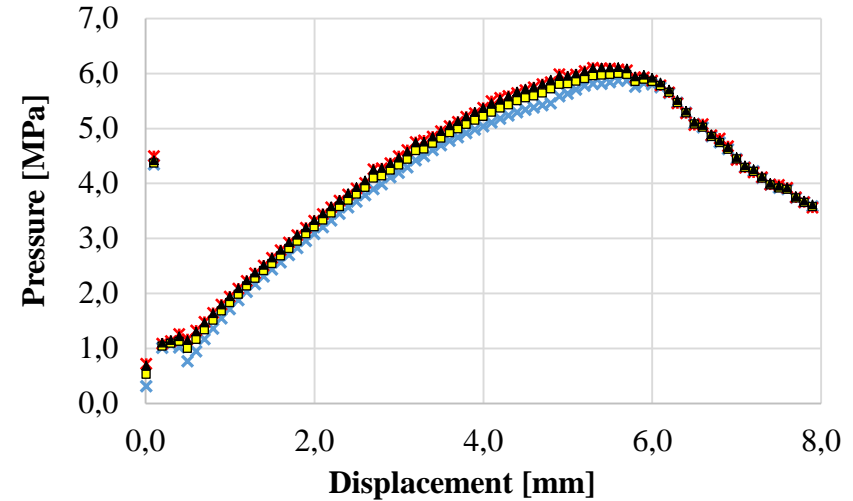


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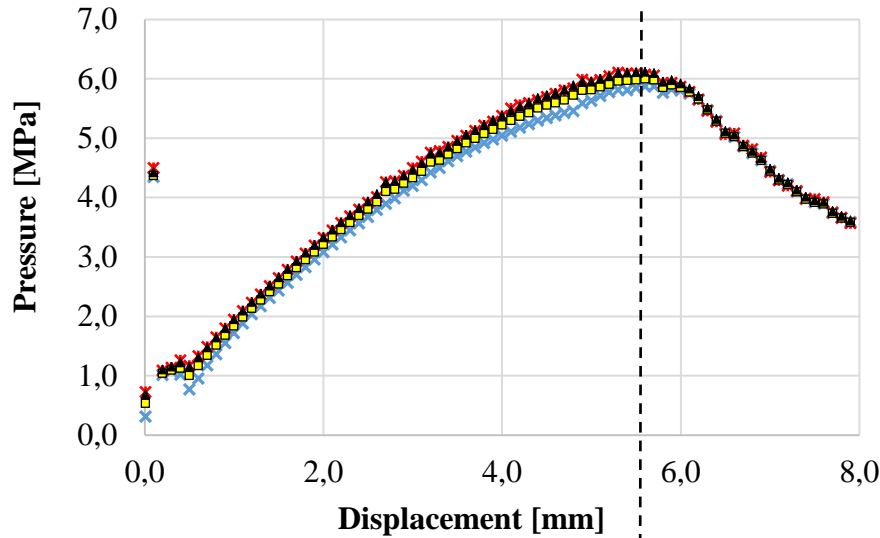
$$\bar{\sigma} = 140\bar{\epsilon}^{0.04} \text{ (MPa)}$$

Fluid: $\mu = 10^2 \text{ Pa} \cdot \text{s}$

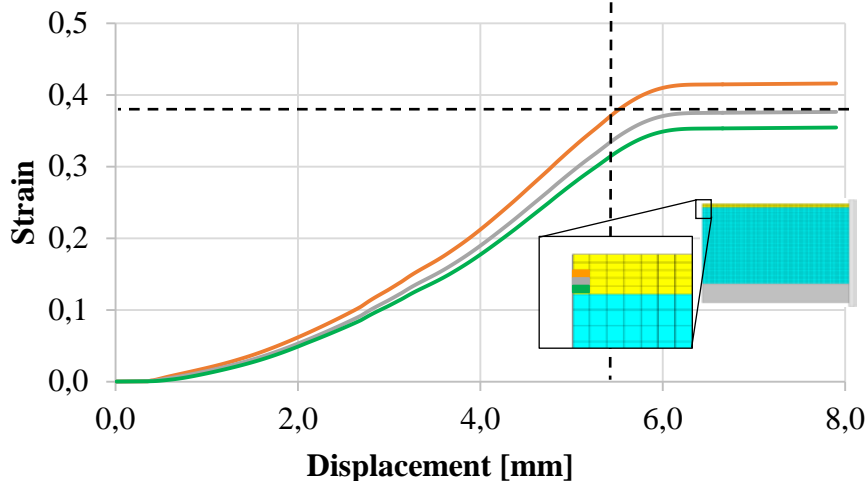
$$\mu_v = 10^3 \text{ MPa} \cdot \text{s}$$



Bulge Test



uniform distribution of hydrostatic pressure!



onset of instability (Hill, 1950)

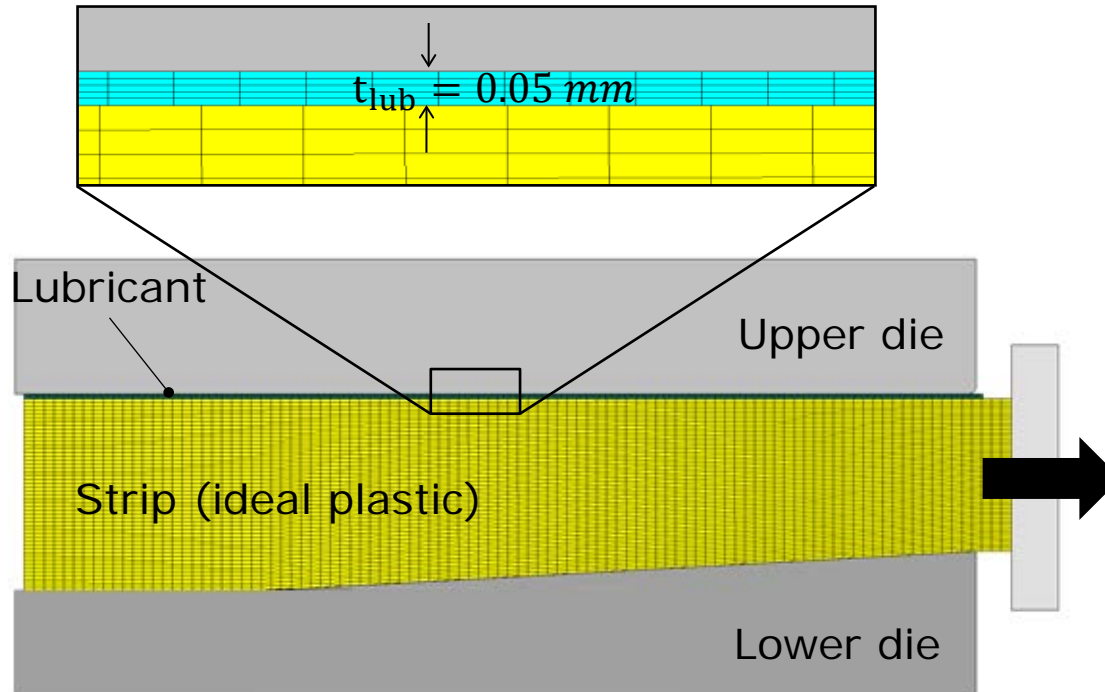
$$\bar{\epsilon}_{inst,cal} = \frac{4}{11}(2n + 1)$$

$$\bar{\epsilon}_{inst,cal} = 0.39$$

$$\bar{\epsilon}_{inst,num} = 0.37$$

error = 2 %

Lubricant Breakdown



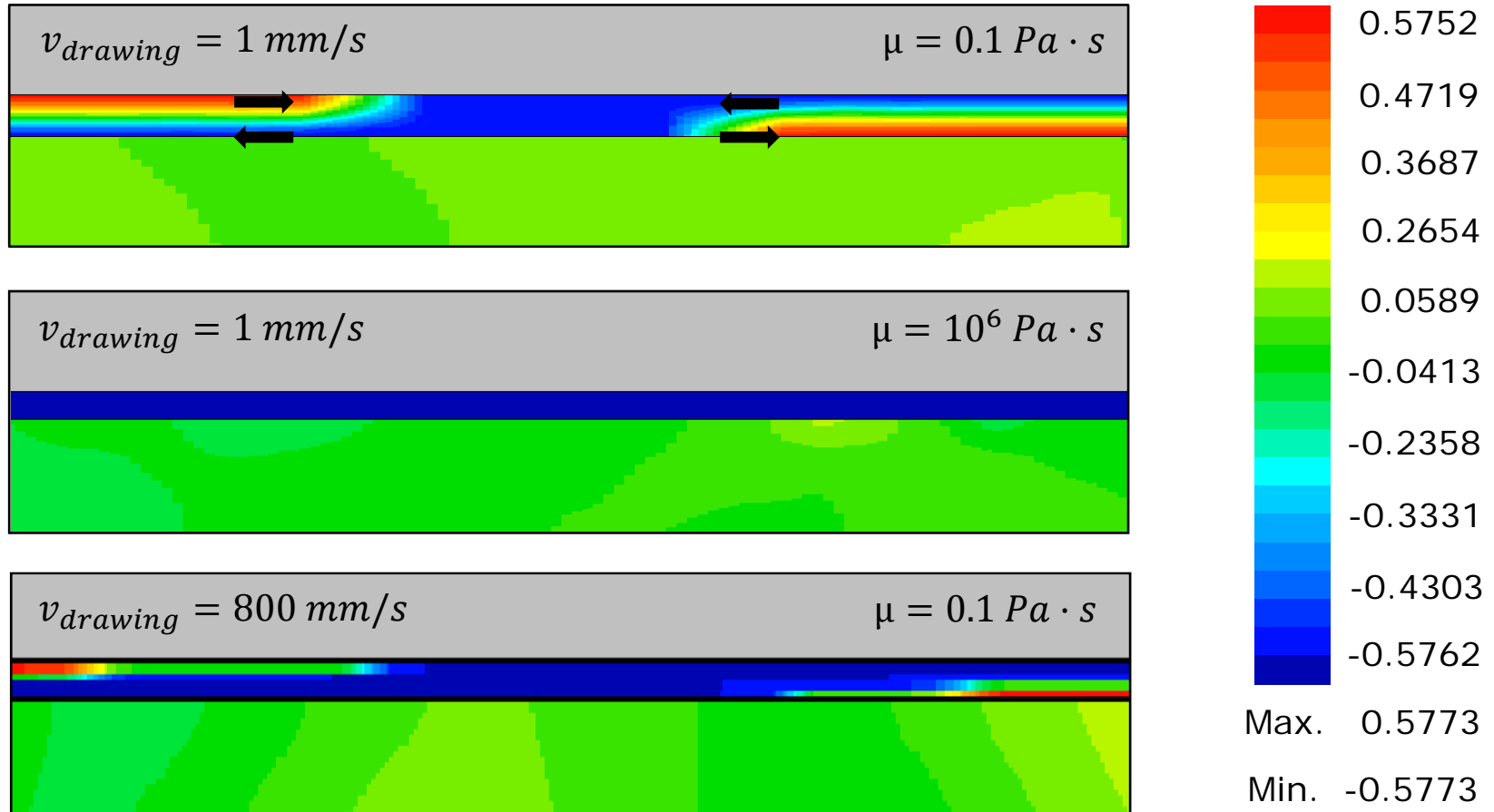
$m = 1.0$ sticking interfaces

$\alpha_{lower \text{ die}} = 3.0^\circ$

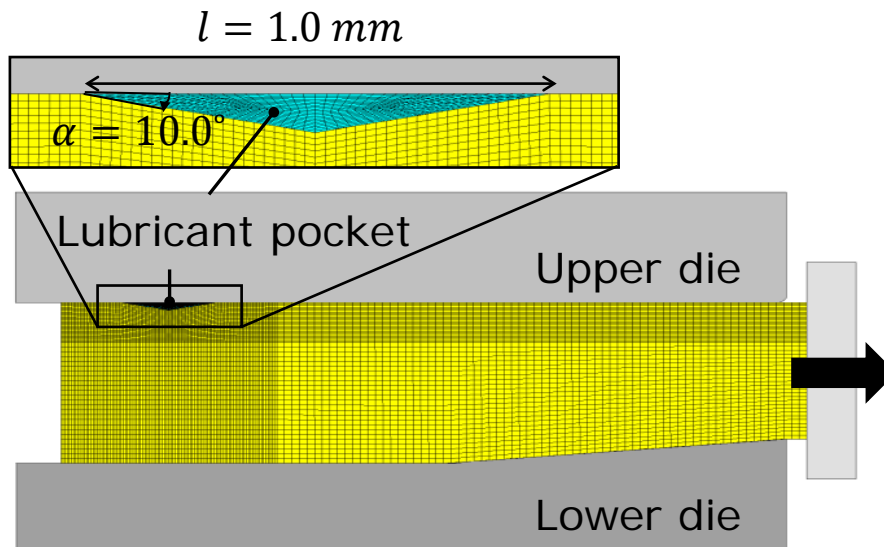
$h_{strip} = 1.95 \text{ mm}$

Lubricant Breakdown

Shear Stress Normalized by Effective Stress



Drawing of Sheet with Surface Pocket



$$v_{drawing} = 100 \text{ mm/s}$$

$$\alpha_{lower die} = 5.0^\circ$$

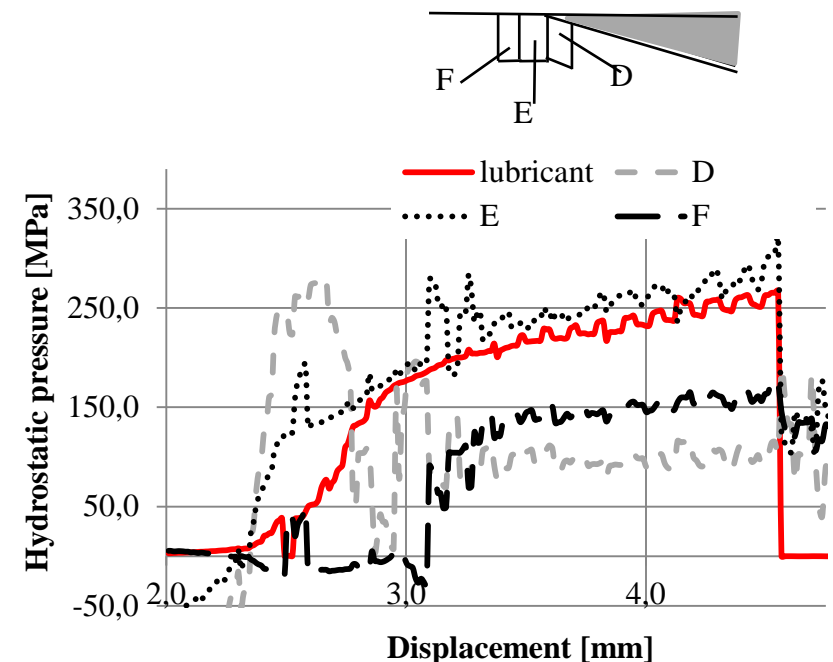
$m = 0.3$ along the upper tool

$m = 1.0$ sheet – lubricant interface

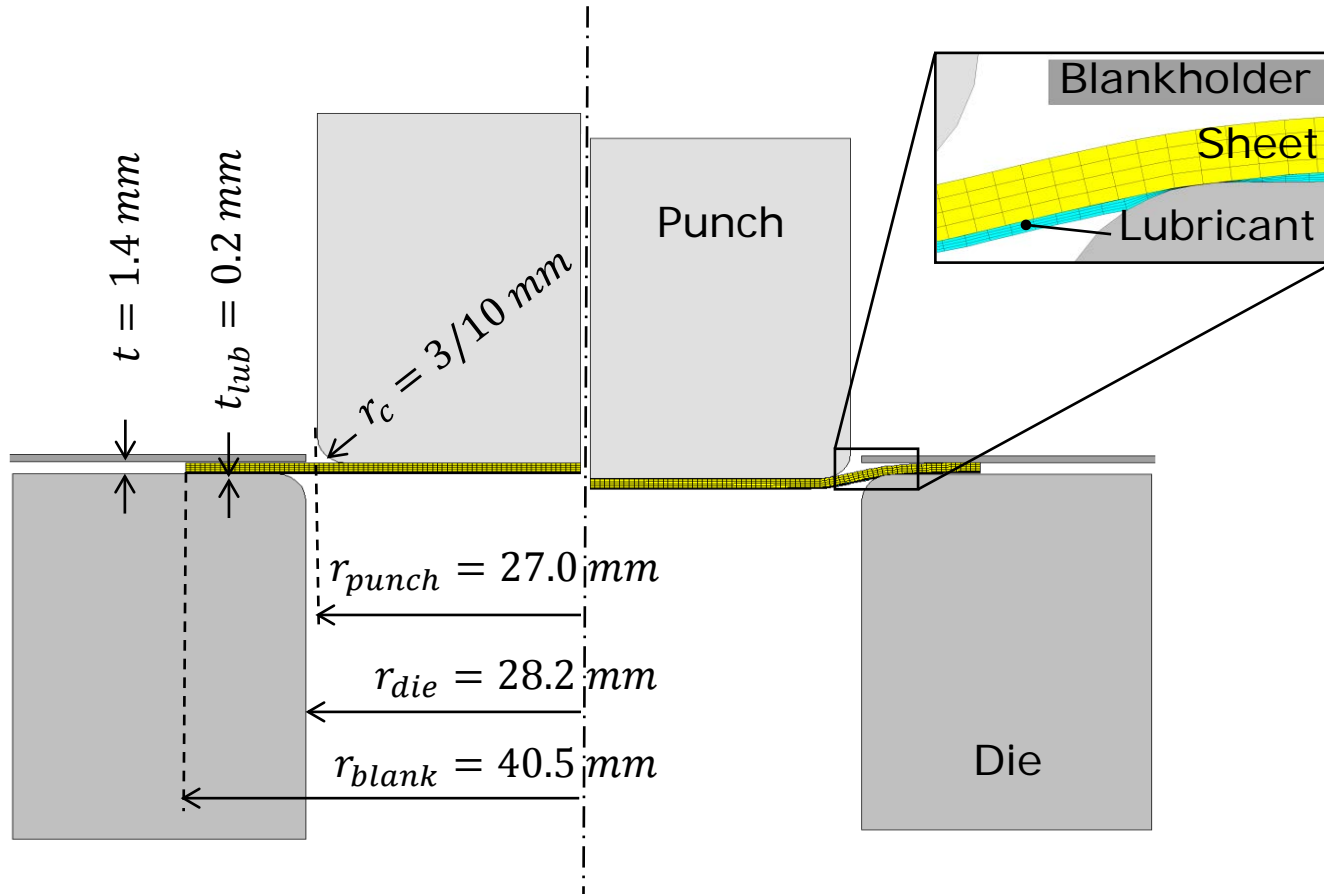
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$$\bar{\sigma} = 312\bar{\epsilon}^{0.08} \text{ (MPa)}$$

Fluid: $\mu = 0.1 \text{ Pa} \cdot \text{s}$



Deep Drawing

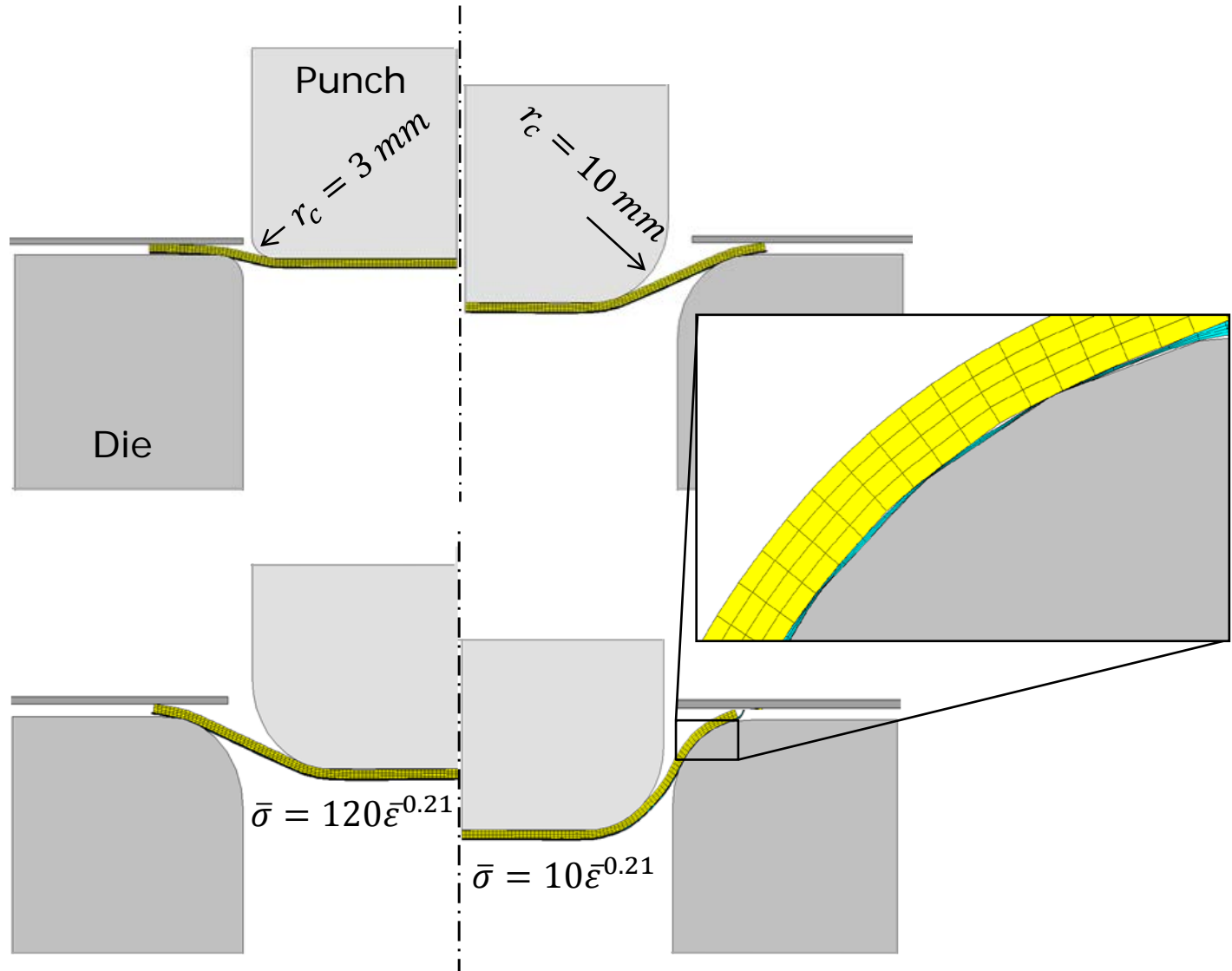


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 Fluid: $\mu = 10^6 \text{ Pa} \cdot \text{s}$

$m = 0.5$ sheet – lubricant interface
 $v_{drawing} = 1 \text{ mm/s}$

Deep Drawing

$$\bar{\sigma} = 120\bar{\epsilon}^{0.21}$$

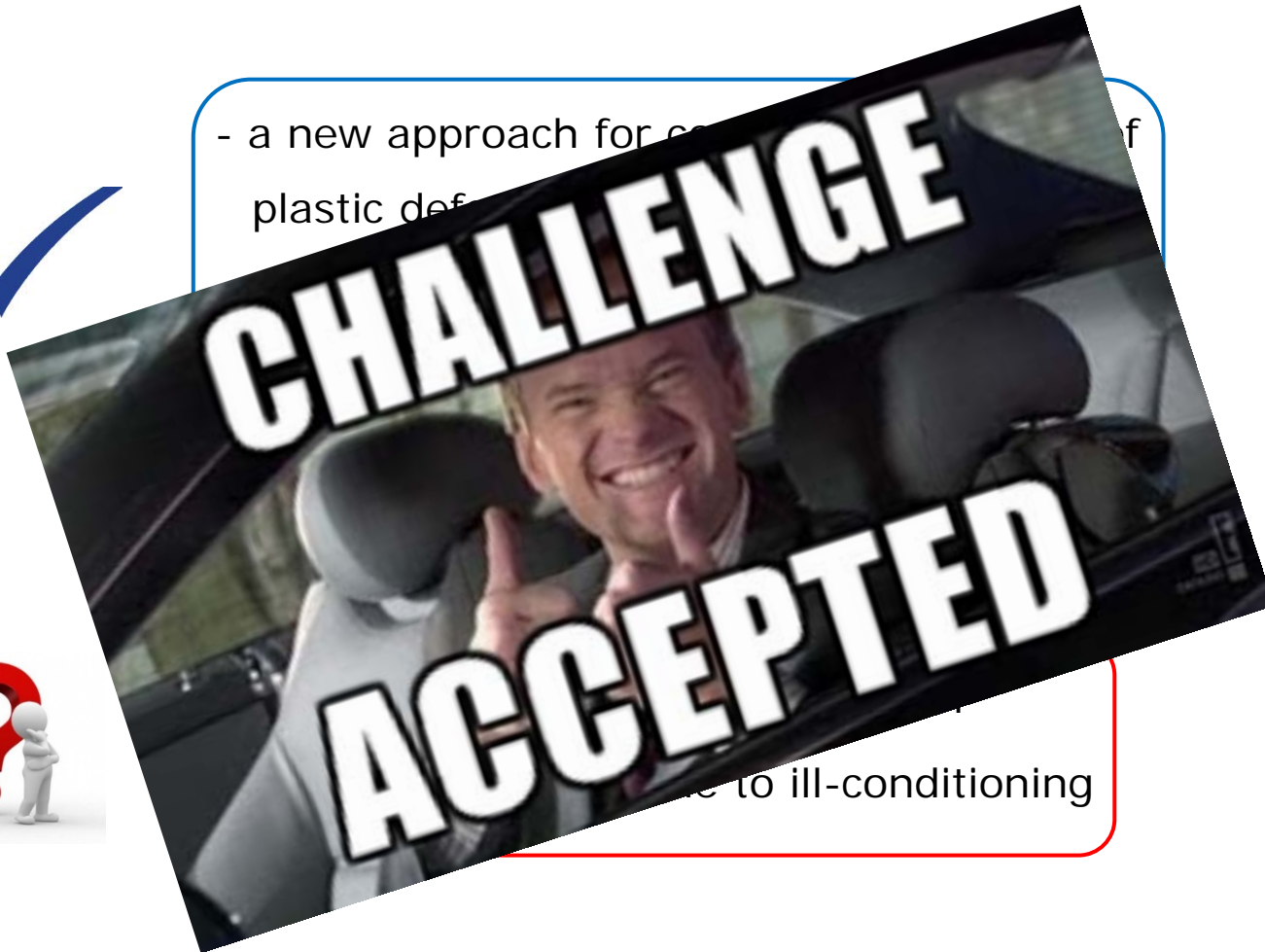


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 - Current State
 - Challenges
 - Future Work

Summary

- a new approach for...
plastic def...



...to ill-conditioning

Future Works



decoupling of the two models simulating
metal flow and liquid flow separately

Back-Up



The Flow Formulation

- The weak form of the quasi static stress equilibrium equations used in the flow formulation results from neglecting gravity and dynamic effects.
- Incompressibility is included by transforming the general weak form of the stress equilibrium equations into a weak form with constraints.
- Flow formulation is based on a velocity-pressure approach where metals are considered as fluids of very high viscosity and velocities and hydrostatic pressures are solved simultaneously.

Drawing of Sheet with Surface Pocket

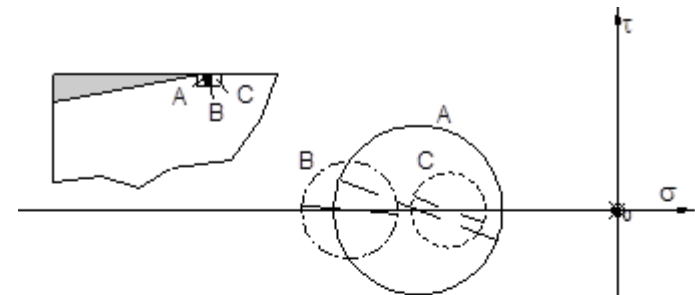
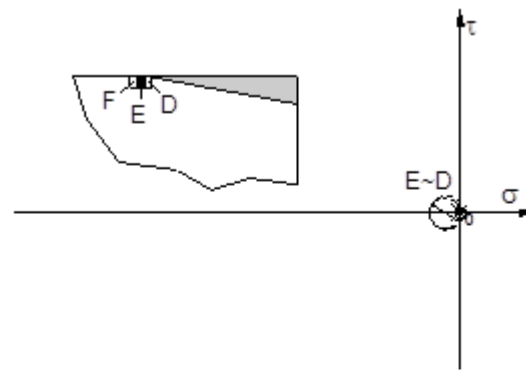
Back of the Pocket

$$v_{drawing} = 100 \text{ mm/s}$$

Front of the Pocket

$$v_{drawing} = 1 \text{ mm/s}$$

Beginning
of the Process



Later Stage

